FDD Applied to a Residential Split System Heat Pump

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Residential split-system heat pump

System Characteristics

- Refrigerant: R410A
- 2.5-ton split heat pump syste
- 13 SEER, 8.5 HSPF
- TXV in cooling and heating m odes
- Scroll compressor





ID unit (Indoor A-coil in upflow)



FDD in the heating mode

- Steady-State Detector (SSD)
- Polynomial Reference Model for important system features
- Fault Detector/Classifier

Cooling	Heating			
Under/Over Charge	Under/Over Charge			
Improper Airflow through HXs	Improper Airflow through HXs			
Liquid Line Restriction	Liquid Line Restriction			
Compressor/Reversing Valve Leakage	Compressor/Reversing Valve Leakage			
Non-condensables				

Fault-free reference model

- FDD scheme requires a reference fault-free value for features
- Real-time sampling of feature values requires a steady-state detector

Independent variables

 T_{OD} : outdoor or condenser air dry-bulb temperature T_{ID} : indoor or evaporator air dry-bulb temperature T_{IDP} : indoor or evaporator air dew-point temperature

Example: Second order multivariate reference polynomial

$$\phi_{i} = a_{0} + a_{1}T_{OD} + a_{2}T_{ID} + a_{3}T_{IDP} + a_{4}T_{OD}T_{ID} + a_{5}T_{ID}T_{IDP} + a_{6}T_{ID}T_{IDP} + a_{6}T_{ID}T_{ID} + a_{6}T_{ID}T$$

All measurements, including test conditions, have uncertainties.

Features and residuals

• Features: Characteristic measurements to detect and diagnose faults



- $T_{\rm ER}$: evaporator exit saturation temperature
- $\Delta T_{\rm sh}$: evaporator exit superheat
 - : compressor discharge wall temperature
 - : condenser inlet saturation temperature
 - : liquid line subcooling
 - : condenser air temperature rise
- $\Delta T_{\rm EA}$: evaporator air temperature drop
- $\Delta T_{\rm LL}$: liquid line temperature drop. \triangleright
- **Residuals:** difference in measurements and expectations
 - Expectations are estimated by no-fault reference model.

Residual,
$$\Delta \phi = \phi_{\text{Measurement}} - \phi_{\text{Reference}}$$

Fault implementation example

Improper outdoor air flow rate (Condenser fouling)

Reduction of air flow

i.e., obstructions stuck to HX, bent fins, fan fault

Fault Implementation:

Blocking the lower part of the outdoor HX.



Partially blocked condensing unit.





Residual pattern for a compressor/reversing valve fault



Probability of a fault

$$z_{i} = \frac{x_{i} - \mu_{i,\text{NF}}}{\sigma_{i}}$$

$$P(C_{\downarrow}, z_{i}) = P(z_{i} \ge s) = \frac{1}{2} \left[1 - \operatorname{erf} \left\{ \frac{1}{\sqrt{2}} (z_{i} + s) \right\} \right]$$

$$P(C_{\uparrow}, z_{i}) = P(z_{i} \le s) = \frac{1}{2} \left[1 + \operatorname{erf} \left\{ \frac{1}{\sqrt{2}} (z_{i} - s) \right\} \right]$$

$$\mathcal{E}_{i} \approx S\sigma_{i}$$

$$P(C_{-}, z_{i}) = P(-s < z_{i} < s) = \frac{1}{2} \left[\operatorname{erf} \left\{ \frac{1}{\sqrt{2}} (z_{i} + s) \right\} - \operatorname{erf} \left\{ \frac{1}{\sqrt{2}} (z_{i} - s) \right\} \right]$$



Cooling rule-based chart a, b, c

Fault Type	T _E	ΔT _{SH}	T _{CMP,D}	T _c	∆T _{sc}	ΔT_{CA}	ΔT_{EA}
Compressor fault	↑	-	_d	\leftarrow	\checkmark	\leftarrow	\checkmark
Condenser fouling	-	↑e	1	<	\checkmark	\rightarrow	\checkmark
Evaporator fouling	\checkmark	-	◆	\checkmark	-	\checkmark	$\mathbf{\uparrow}$
Liquid line restriction	\rightarrow	1	◆	←	1	\rightarrow	$\mathbf{\mathbf{\mathbf{\mathbf{\mathbf{\mathbf{\mathbf{\mathbf{\mathbf{\mathbf{\mathbf{\mathbf{\mathbf{\mathbf{\mathbf{\mathbf{\mathbf{\mathbf{$
Refrigerant overcharge	-	\rightarrow	<	←	1	-	-
Refrigerant undercharge	\rightarrow	4	<	\rightarrow	\checkmark	\rightarrow	\mathbf{A}
Non-condensable gases	-	-	1	1	$\mathbf{\uparrow}$	-	-
No fault	-	-	-	-	-	-	-

^a Residential split air conditioner with a constant speed scroll compressor and a TXV.

^b \uparrow , \downarrow , - represent the direction of positive, negative, and neutral residual respectively.

^c No change implies the variations within the uncertainty of measurement.

^{d,e} Cells colored by light blue and pink show inconsistent rules compared to the result of Che n and Braun (2001)

Heating rule-based chart

Fault Type	T _E	ΔT _{sh}	T _{CMP,D}	T _c	ΔT _{sc}	ΔT_{CA}	ΔT_{EA}
Compressor fault	- 个	-	-	\rightarrow	-	- 🗸	-
Condenser fouling	-	-	↑	- 个	-	\uparrow	-
Evaporator fouling	\checkmark	¥	^-↓	- 🗸	-	- 🗸	-
Liquid line restriction	-	-	-	-	-	-	-
Refrigerant overcharge	-	-	↑	$\mathbf{\uparrow}$	1	-	-
Refrigerant undercharge	- 🗸	-	-	- 🗸	\checkmark	- 🗸	-
No fault	-	-	-	-	-	-	-

Cooling/Heating rule-based chart comparison

Fault Type	Τ _Ε	$\Delta {\rm T}_{\rm SH}$	T _{CMP,D}	Т _с	$\Delta \mathbf{T_{sc}}$	$\Delta \mathbf{T}_{CA}$	$\Delta \mathbf{T}_{\mathrm{EA}}$
Compressor fault	↑ - ↑	-	-	\downarrow	↓ -	↓ -↓	→ -
Condenser fouling	-	↑ .	\uparrow	↑ - ↑	↑	\checkmark \uparrow	→ -
Evaporator fouling	\checkmark	-	$\uparrow \qquad \qquad$	↓ - ↓	-	↓ -↓	← -
Liquid line restriction	→ -	↑.	↑	↑	↑	↓ .	→ -
Refrigerant overcharge	-	→ -	\uparrow	\uparrow	\uparrow	-	-
Refrigerant undercharge	↓ -↓	↑	↑	↓ -↓	\downarrow	↓ -↓	≁ -
No fault	-	-	-	-	-	-	-

Fault classifier module

🔁 FDD Main Offline 02.vi _ 🗆 🗡 File Edit Operate Tools Browse Window Help 2 2 🔿 🕹 🔵 🗉 Fault Detection and Diagnostic System for Split Heat Pump Fault Classifier EER Degradation (%) -Minsung Kim, HVAC&R Equipment Performance Group 0.2--2.59318 -1.0 Papility 0.1 - 0.05 -Alarm Control (%) Moving Window Moving Avg -2 100.056 100.067 100.103 99.9717 100.018 100.043 Tcai EER Warning 0-1 69.9728 Teai 69.9515 69.9901 69.8903 69.9787 70.0535 1 1 1 1 1 CMF CF EF LL OC UC NC NF 50.445 Teai_dp 50.4155 50.4155 50.4647 50.4647 50.4647 Moving Avg Out Model Out Residual Standard Deviation Probability 42.4071 42.5109 42.5109 42.4849 42.5109 44.3176 0.370591 0 TE 42.4849 -1.8326712.7436 0.276176 0.819581 0 12.547 12.5324 12.5841 12.8253 12.6465 12.3703 dsh 181.404 1.8619 1.08599 0.156413 183.447 183.205 183.341 183.266 183.128 183.208 TD 117.577 118.065 -0.487064 0.232733 0 117.519 117.609 117.631 117.564 117.564 TC 6.91052 -0.409082 0.474247 3.81497E-31 6.50144 6.5358 6.59834 6.47791 6.39059 6.50457 dsc 14.8991 -0.305276 0.197122 0 14.5938 14.5799 DTca 14.5838 14.5209 14.6438 14.6408 17.881 2.22564 0.383406 2.33688E-29 20.1066 20.1359 20.133 20.021 20.1239 20.1193 DTea 5.92341E-28 EER Residual EER Stdev Model Out EER Moving Avg -0.271364 0.391699 10.4645 EER 10.2042 10.1869 10.1854 10.1788 10.2105 10.1932 NIST

Example of a FDD System with 15% loss in evaporator airflow

What more do we need?

- Commissioning tool using FDD module
- Learning module
 - System adjusts to every installation
 - System adjusts as it ages
- Heating mode unique problems
 - Refrigerant charging in heating mode
 - Frosting of the outdoor heat exchanger

What more do we need? (cont.)

- Standardized fault codes for residential systems
 - Everyone needs to define a fault in the same way
 - Standard fault codes simplify communicating faults to a central system
- FDD for variable speed systems and multifunction/hybrid systems
- FDD for mixed systems
- Standard method for comparing FDD devices/algorithms

List of FDD publications from BFRL

- 2009, "Heating Mode Performance Measurements for a Residential Heat Pump with Single Faults Imposed," NIST Technical Note 1648, National Institute of Standards and Technology, Gaithersburg, MD.
- 2009, "Performance of a Residential Heat Pump Operating in the Cooling Mode with Single Faults Imposed," Applied Thermal Engineering 29(4), 770-778.
- 2008, "Cooling Mode Fault Detection and Diagnosis Method for a Residential Heat Pump," NIST Special Publication 1087, National Institute of Standards and Technology, Gaithersburg, MD.
- 2008, "Design of a steady-state detector for fault detection and diagnosis of a residential air conditioner," International Journal of Refrigeration 31(5), 790-99.
- 2006, "Performance of a Residential Air Conditioner at Single-Fault and Multiple-Fault Conditions," NISTIR 7350, National Institute of Standards and Technology, Gaithersburg, MD.