



# HCCI and Stratified-Charge CI Engine Combustion Research

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**U.S. DOE, Office of Vehicle Technologies**  
**Annual Merit Review and Peer Evaluation**



**Program Manager: Gurpreet Singh**

**Project ID: ace\_04\_dec**

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# Overview

## Timeline

- Project provides fundamental research to support DOE/Industry advanced engine projects.
- Project directions and continuation are evaluated annually.

## Budget

- Project funded by DOE/VT:  
FY08 – \$695k  
FY09 – \$700k

## Barriers

- Extend HCCI (LTC) operating range to higher loads.
- Improved understanding of in-cylinder processes.
- Control HC & CO emiss. at low loads.

## Partners / Collaborators

- Project Lead: Sandia  $\Rightarrow$  John E. Dec
- Part of Advanced Engine Combustion (AEC) working group:
  - 15 Industrial partners: auto, engine & energy
  - 5 National Labs & Univ. of Wisconsin
- GM – bimonthly meetings & discussion
- Chevron – funds complementary project
- LLNL – 1) support kinetic-mechanism devel., 2) CFD modeling, & 3) cooperative project on detailed exhaust speciation.



# Objectives

**Project objective: to provide the fundamental understanding (science-base) required to overcome the technical barriers to the development of practical HCCI and HCCI-like engines by industry.**

## **FY09 Objectives:**

- Determine the development of natural thermal stratification in an HCCI engine, using planar-imaging thermometry.
  - Thermal-imaging diagnostic developed as part of this task.
- Evaluate the potential of intake boost for extending the high-load limit of HCCI by using EGR to control combst.-phasing advance – multi-year task.
  - FY09: Determine potential of boost with EGR for gasoline at rep. engine speed.
- Determine the performance of ethanol as a fuel for HCCI engines.
  - Conducted cooperatively with M. Sjöberg in the Advanced SI-Engine Fuels Lab.
- Support CFD modeling and the development/improvement of chemical-kinetic mechanisms for HCCI at LLNL  $\Rightarrow$  provide data and analysis.

# Milestones

## FY2008

- Complete analysis of detailed exhaust-gas speciation measurements for iso-octane. (February 2008) – Status: Completed
- Determine the potential benefits of EGR for reducing the maximum pressure-rise rate and extending the high-load HCCI limit. (August 2008) – Status: Completed

## FY2009

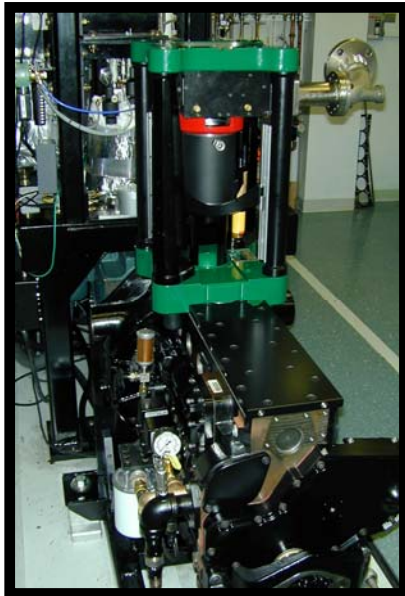
- Determine the magnitude and distribution of the natural thermal stratification in an HCCI engine at a typical operating condition. (February 2009) – Status: Completed.
- Determine the potential of EGR for increasing the allowable intake-pressure boost for gasoline-like fuels, at a representative engine speed. (August 2009) – Status: ~60% complete as of March 2009.

# Approach

- Use a combination of metal- and optical-engine experiments and modeling to build a comprehensive understanding of HCCI processes.
- Metal engine  $\Rightarrow$  design well-characterized experiments to isolate specific aspects of HCCI/SCCI combustion & relationships between parameters.
  - Intake boost: Systematically increase boost  $\Rightarrow$  adjust  $T_{in}$  and/or EGR to retard timing to allow max. fueling at each  $P_{in}$  without knock, but with good stability.
- Optical engine  $\Rightarrow$  detailed investigations of in-cylinder processes.
  - Thermal stratification (TS): Develop temperature-imaging diagnostic  $\Rightarrow$  Apply to obtain T-map images showing temporal and spatial development of TS.
- Computational Modeling  $\Rightarrow$  supplement experiments by showing cause-and-effect relationships that are not easily measured.
  - Initiating LES modeling with J. Oefelein, Sandia to understand mechanism of TS.
  - In-house CHEMKIN (Senkin) single- and multi-zone kinetic modeling.
  - Collaborate with LLNL to improve kinetic mechanisms, and on CFD modeling.
- Combination of techniques provides a more complete understanding.
- Transfer results to industry.

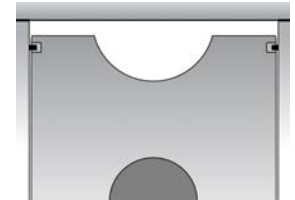
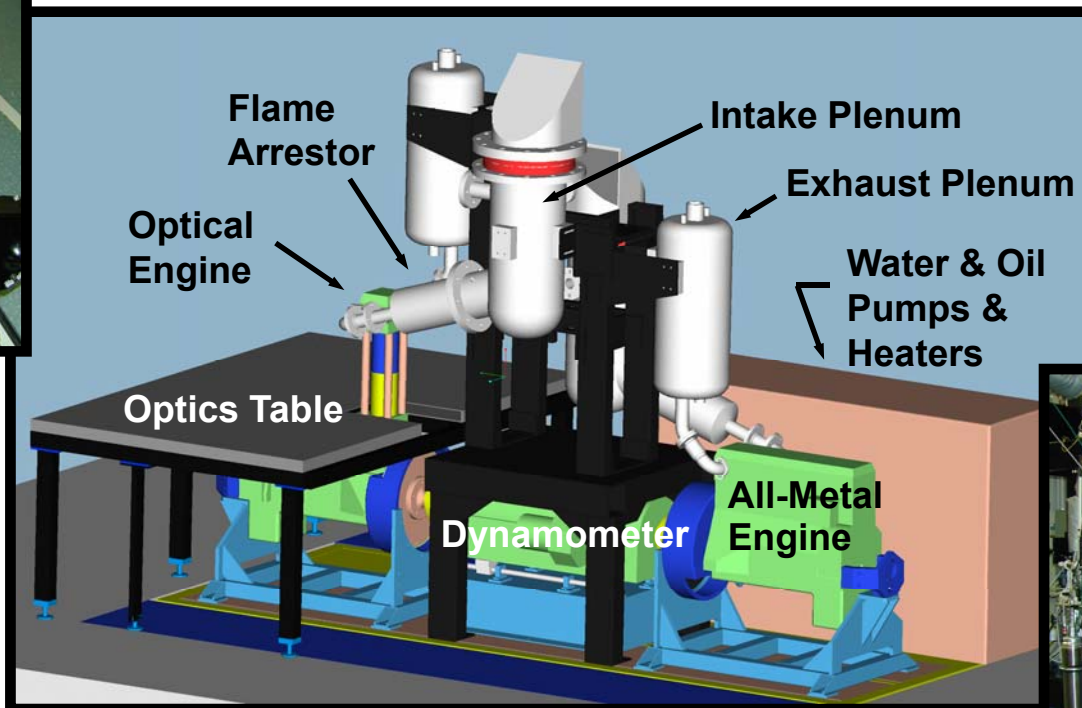


# Sandia HCCI / SCCI Engine Laboratory

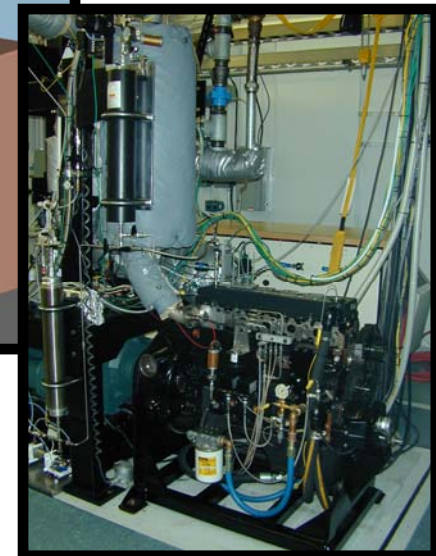


**Optical Engine**

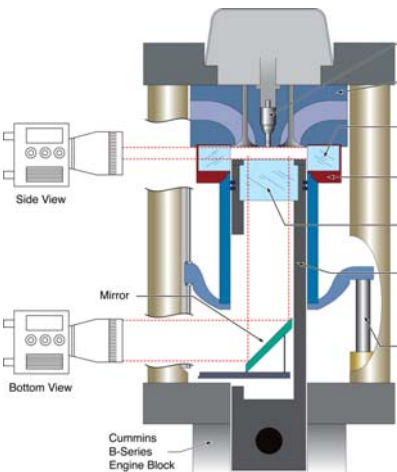
- Matching all-metal & optical HCCI research engines.
  - Single-cylinder conversion from Cummins B-series diesel.



**All-Metal Engine**



- Bore x Stroke = 102 x 120 mm
- 0.98 liters, CR=14





# Accomplishments

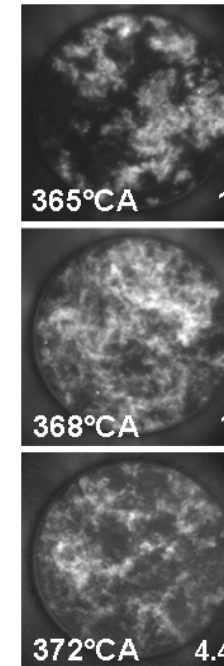
- Determined the evolution of natural thermal stratification in an HCCI engine, including its distribution and magnitude at a typical operating condition.
  - Developed a planar temp.-imaging diagnostic for TS in HCCI engines.
- Conducted initial investigation showing the potential of intake boost for extending the high-load limit of HCCI for gasoline fuel.
  - Showed that EGR is effective for controlling boost-induced timing advance.
  - Achieved a substantial load increase at a rep. 1200 rpm operating condition.
- Determined the behavior of ethanol as an HCCI fuel over a range of operating conditions.
  - Cooperatively with M. Sjöberg of the Advanced SI-Engine Fuels Lab.
- Initiated detailed exhaust-speciation analysis for PRF80  $\Rightarrow$  2-stage ignition.
  - Project conducted in cooperation L. Davisson at LLNL.
- Supported chemical-kinetic and CFD modeling work at LLNL.
  - Provided data and analysis for: 1) improving chemical-kinetic mechanisms, and 2) CFD modeling of fuel stratification to improve low-load comb. eff. & emissions.

# Importance of Thermal Stratification (TS)

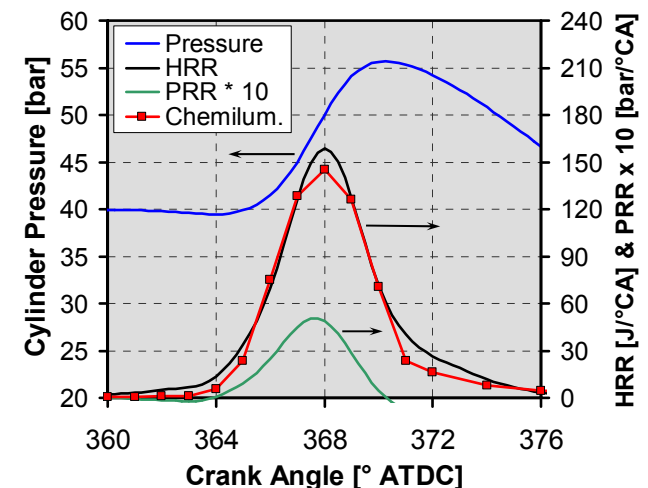
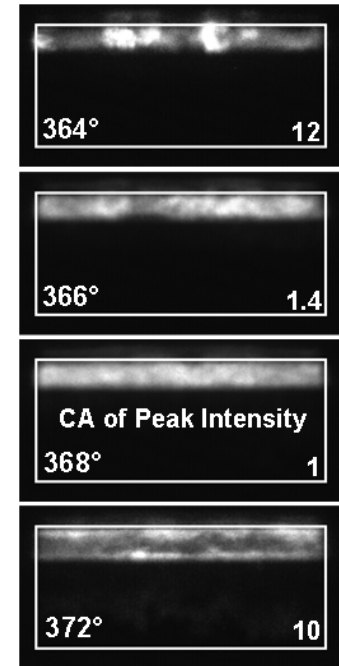
- TS causes autoignition to occur sequentially from hottest region to coldest.
  - Reduces max. pressure-rise rate (PRR).
  - Allows higher fueling without knock.
- Amplify the benefit of the TS by retarding combust. timing  $\Rightarrow$  further increases in load.
- Chemilum. images show:
  - Non-uniformities over whole field of view.
  - Hot reactions start intermittently near the mid-plane.
  - At time of max. PRR most combustion is from bulk gases (central region).
  - BL combust. occurs after max. PRR.

- TS of the bulk gas is critical for high-load HCCI operation.
- Understanding TS is important for increasing the high-load limit of HCCI.

**Bottom-View**



**Side-View**

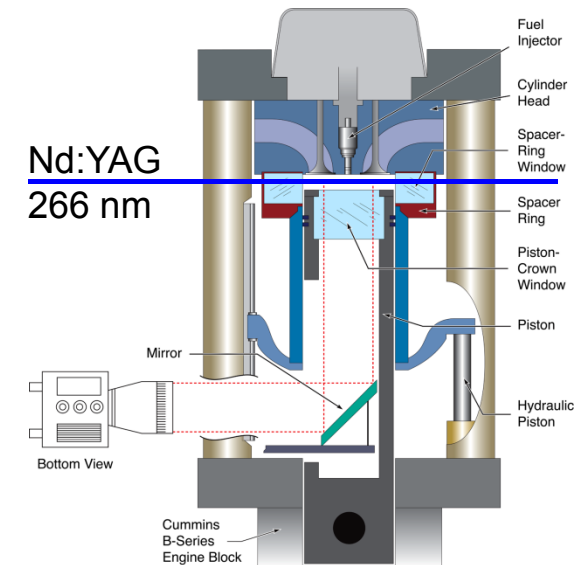




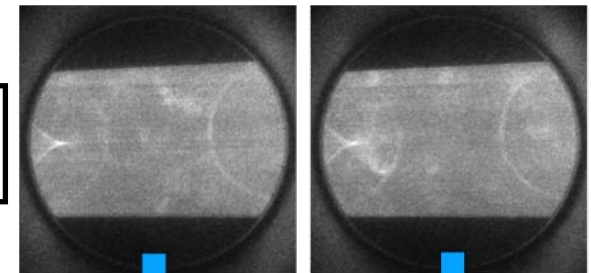


# Planar Imaging Thermometry

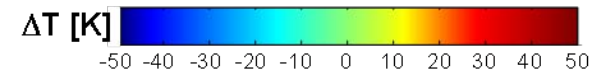
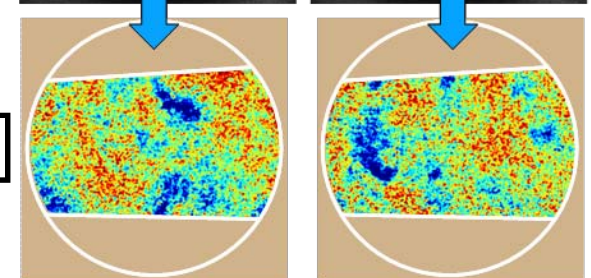
- Diagnostic: single-laser toluene PLIF.
  - PLIF intensity varies with temperature.
  - Good sensitivity in desired range, 600 – 1050 K.
- PLIF setup:
  - 2% toluene + 98% iso-octane
  - Laser excitation: 266 nm, 58 mm wide sheet.
  - Intensified camera with 277nm LP & UG5 filters.
  - Run inert with N<sub>2</sub> to prevent quenching.
    - ⇒ OK since TS develops prior to combustion.
- Calibrate temp. sensitivity in-cylinder.
- For well-mixed fueling, variations in PLIF intensity correspond to temp. variation
  - Temperature fluctuations shown relative to the mean of each image.
  - Bright regions in raw image correspond to cold pockets.



Raw  
PLIF



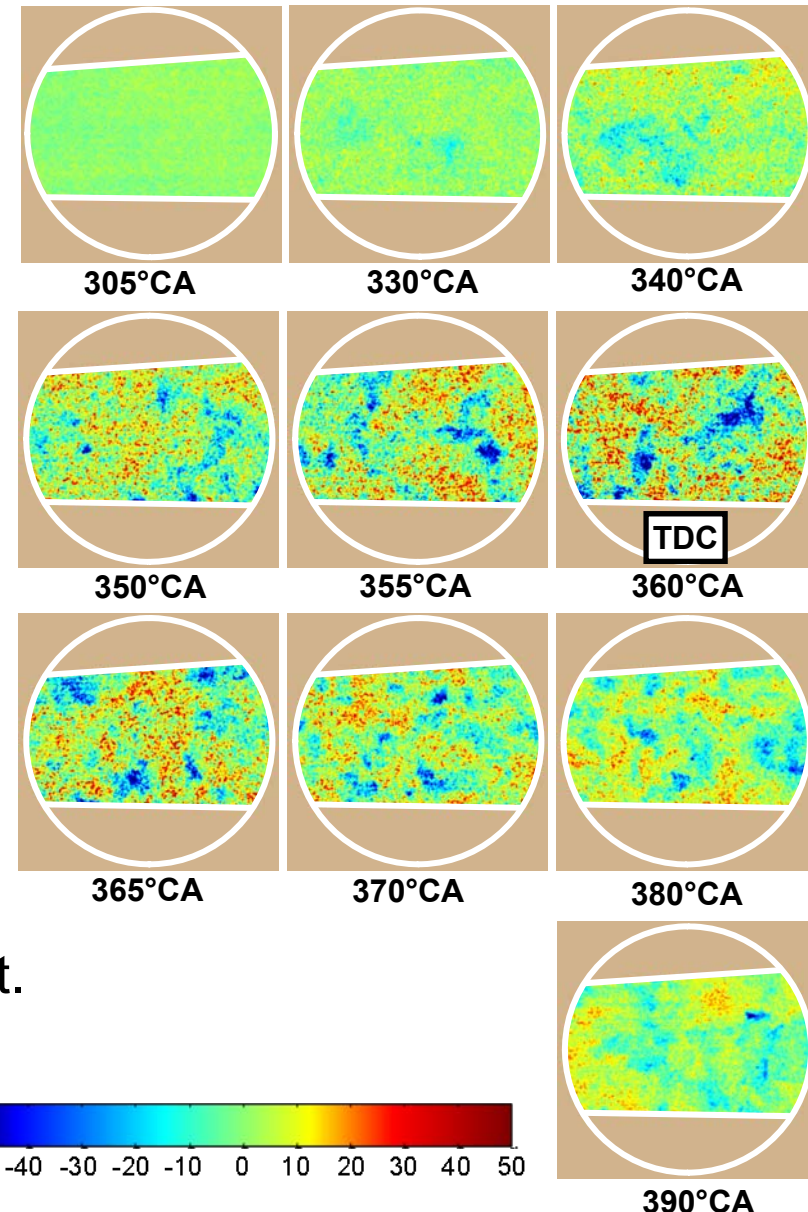
T-map





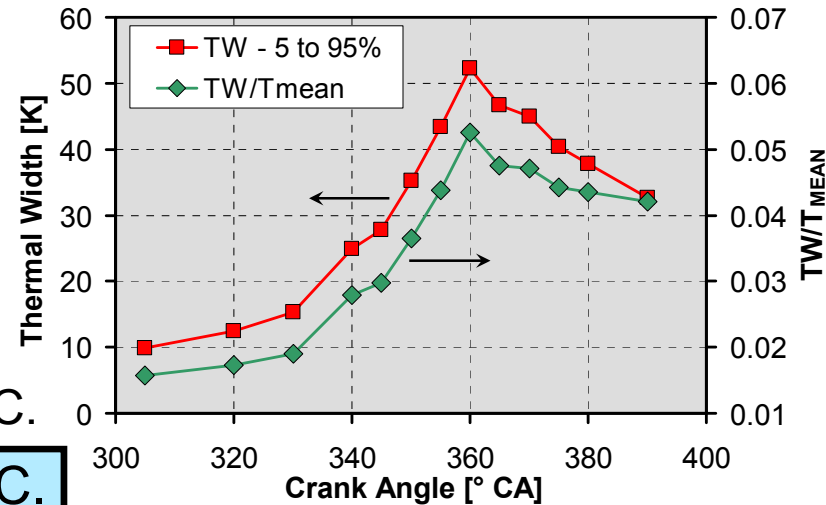
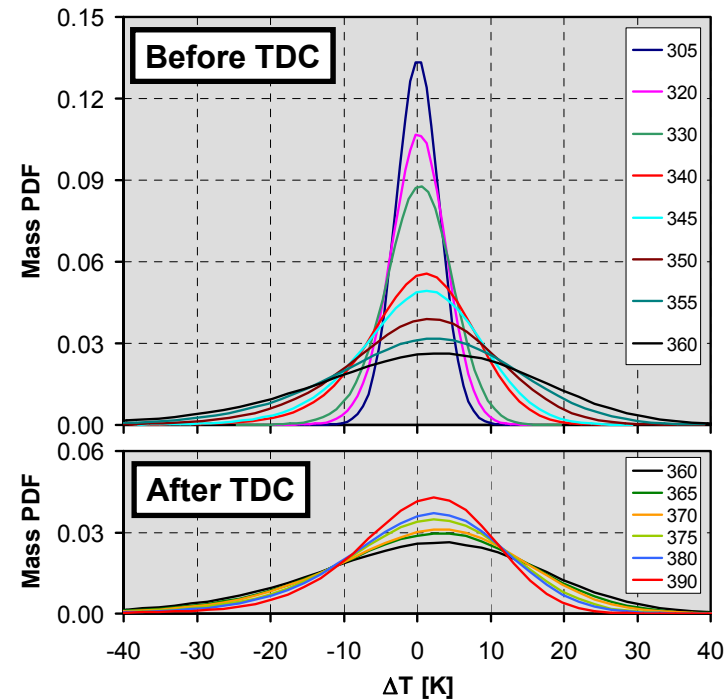
# Temporal Evolution of TS

- Laser elevation adjusted with crank angle to remain in mid-plane (20 - 4 mm below F-D).
  - Representative of bulk gas.
- TS develops progressively as cold pockets convected into central region.
  - Temperature nearly uniform at 305°CA.
    - > Virtually no TS remains from intake.
    - > Insufficient time with  $T_{\text{gas}} > T_{\text{wall}}$ .
  - Substantial TS by TDC (360°CA) +/- 35 K.
    - > Sufficient for significant spread in autoignition time of various regions.
- TS distribution is random cycle-to-cycle.
- Scale of cold pockets near TDC is 5–11 mm, similar to 8mm TDC clearance height. (Fine-grain speckle pattern is shot noise.)
- Magnitude of TS appears to diminish after TDC.



# PDF Analysis of TS Images

- Apply probability density functions (PDF) to quantify changes in temp. distribution with crank angle.
  - 305°CA: PDF very narrow.
    - > Little time for development of TS.
    - > Analysis  $\Rightarrow$  almost all width is shot noise.
  - 330 - 340°CA: Significant broadening.
  - 340 - 360°CA: Progressive increase.
  - 360 - 390°CA: PDF width decreases, in agreement with images.  $\Rightarrow$  cause?
- Define: thermal width (TW) = 5 - 95% of PDF width.
  - Max. TW at TDC  $\approx 50$  K  $\Rightarrow$  agrees with multi-zone model results.
- Normalize by  $T_{\text{MEAN}}$  to remove effects of compression & expansion.
  - Reduces, but not eliminate TW decrs. ATDC.
- Heat transfer dominant  $\Rightarrow$  changes at TDC.





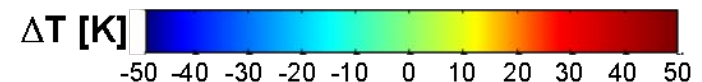
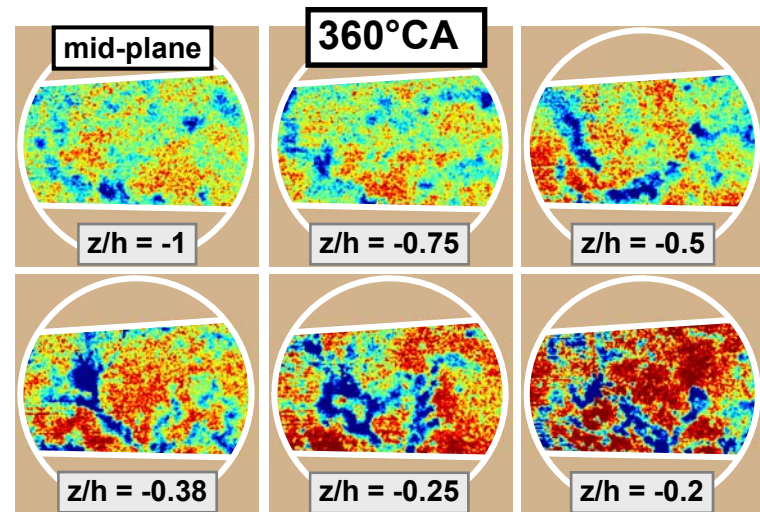
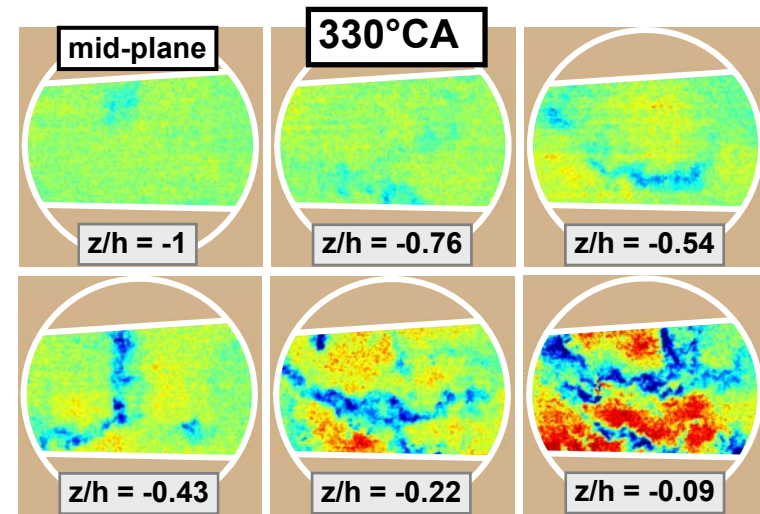
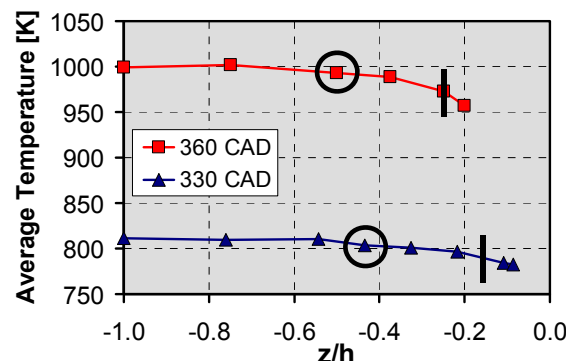
# Thermal Distribution of Boundary Layer (BL)

- Incrementally scan laser from mid-plane to outer BL (to 0.8 mm below firedeck).
- 330°CA: bulk-gas temp. is nearly uniform.  
⇒ Significant TS only for  $z/h \geq -0.43$ .
- 360°CA: TS developed throughout bulk gas.  
⇒ TS greater in outer BL,  $z/h \geq -0.5$ .
- Avg. T profiles also show deficits for these  $z/h$ .

- TS progresses inward from wall,  $330^\circ \rightarrow 360^\circ$
- Most BL temp. deficit occurs in last 0.8 mm at the wall ⇒ drops to  $T_{\text{wall}} \approx 400$  K.

- BL thickness based on a 5% deficit from centerline value  
⇒ 1-1.5mm.

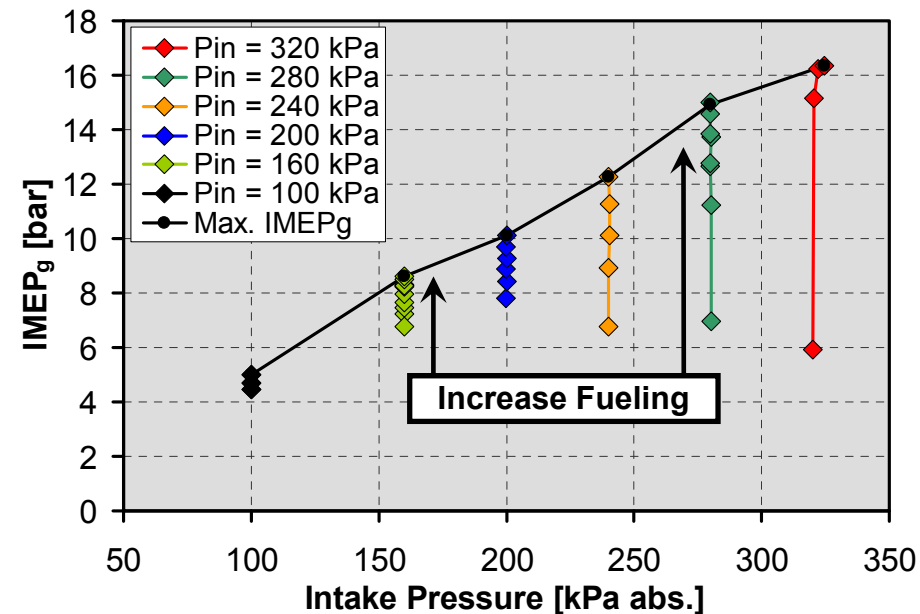
- Agrees with previous chemilum. & PLIF studies.



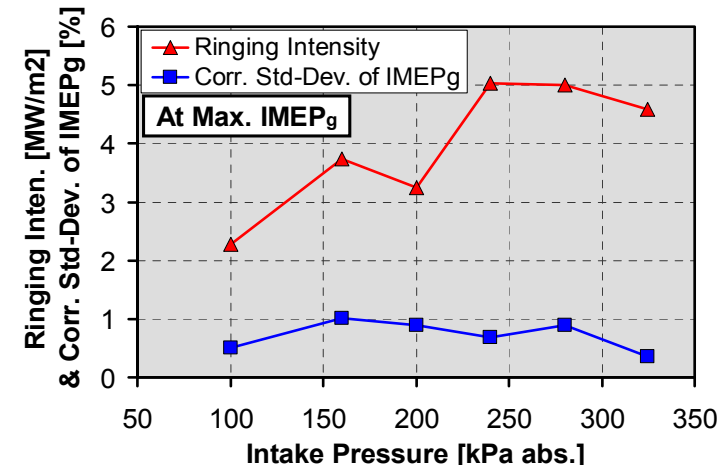


# Intake Boost for Extending High-Load Limit

- Investigate the potential of boosting for extending HCCI to higher-loads.
  - Required to match full-load diesel or SI.
- Current work: gasoline, 1200 rpm.
- Boost enhances autoignition  $\Rightarrow$  advances comb. timing  $\Rightarrow$  Knock!
  - Compensate with reduced  $T_{in}$ .
  - For  $P_{in} > 160$  kPa,  $T_{in} \rightarrow T_{amb}$   $\Rightarrow$  limits allowable fueling.
- Add cooled EGR to further slow autoignition.



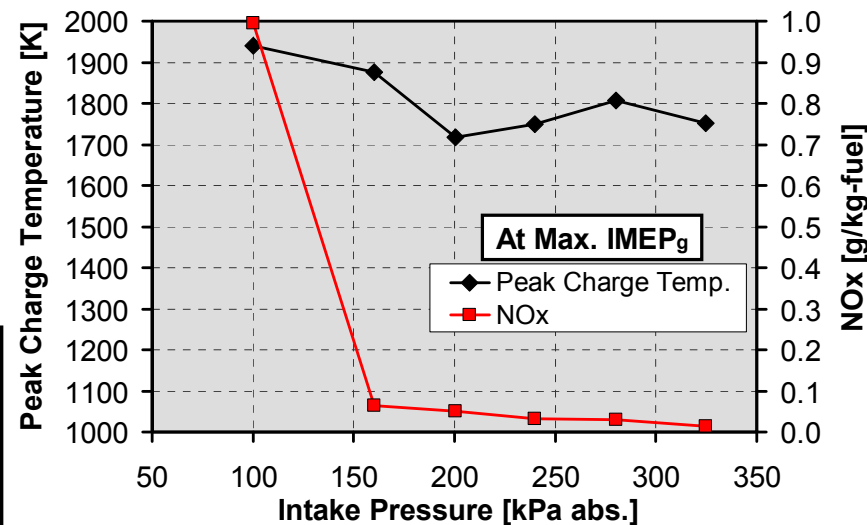
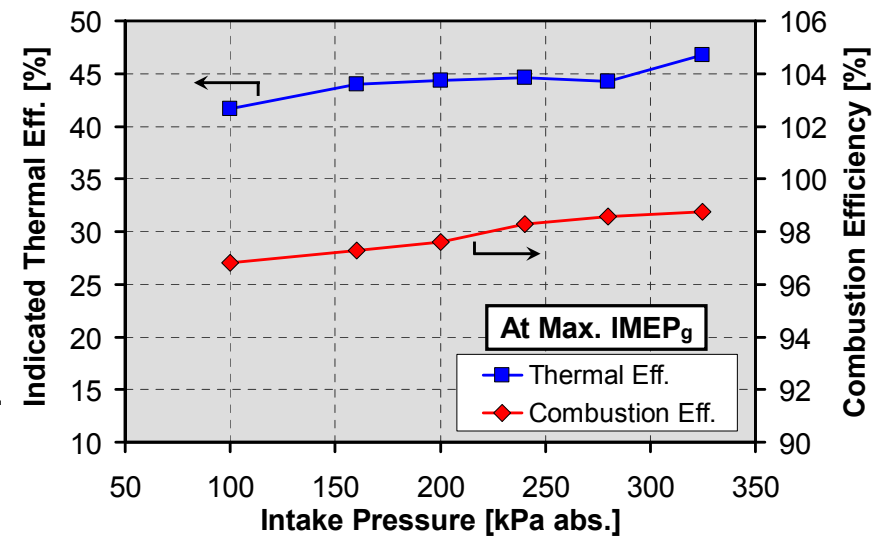
- Achieved **IMEP<sub>g</sub> = 16.3 bar**,  $P_{in} = 324$  kPa.
  - Very high IMEP<sub>g</sub> for HCCI/LTC, convent'l fuel.
  - Near stoich., C/F = 38.5, EGR = 60%,  
 $P_{exhaust} = 326$  kPa,  $T_{exhaust} = 407^\circ\text{C}$ .
- Ringings  $\leq 5$  MW/m<sup>2</sup>, No Knocking.
- Std-Dev of IMEP<sub>g</sub>  $\leq 1\%$ , very good stability.



# Efficiency and NOx for Boosted High-Load

## For maximum IMEP<sub>g</sub> at each boost

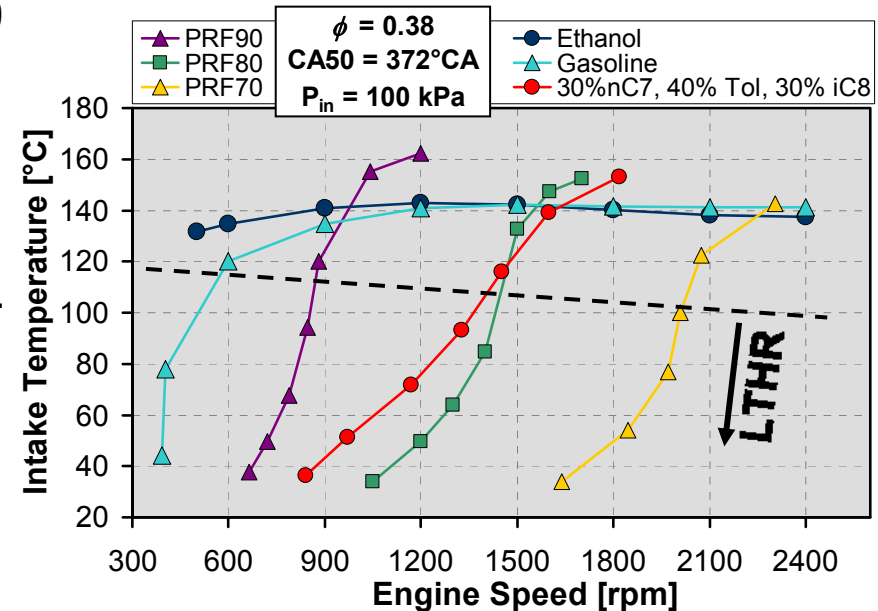
- Indicated Thermal Eff. increases slightly with boost.  $\Rightarrow$  Th. Eff.  $\sim 45\%$ .
  - Combustion Eff. increases,  $97 \rightarrow 99\%$ .
    - Higher wall temps.  $\Rightarrow$  improve combst.
    - Increased EGR reduces HC & CO emiss.
  - NOx emissions below US-2010 stds. (should also meet tier II, bin 5).
    - Extremely low for all boosted cases ( $< 0.1$  g/kg-fuel,  $\sim 1$ -2 ppm).
  - Correlates with low peak charge temp.
    - NOx higher for  $P_{in} = 100$ ,  $T_{peak} > 1900$ K.
- For max. IMEP<sub>g</sub> = 16.3 bar,  $P_{in} = 3.2$  bar.
    - Ind. Thermal Eff. = 47%
    - Comb. Eff. = 99%
    - NOx = 0.015 g/kg-fuel,  $T_{peak} = 1750$  K.



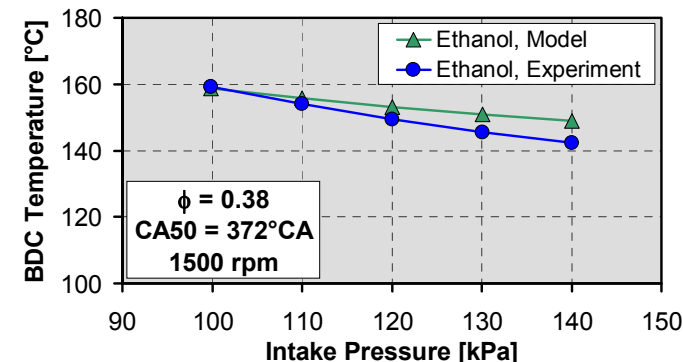


# Ethanol as an HCCI Fuel

- Ethanol is a component in most pump gasoline (0 – 15% fraction).
  - Also being considered as an alternative fuel at levels up to 85%.
- Important to understand ethanol's potential for HCCI.
  - Ignition quality  $\Rightarrow$  RON = 107, MON = 89
  - Effect on performance and operating range, *i.e.* speeds, boost, etc.
- Speed sweep  $\Rightarrow$  CA50 = 372°CA
  - Autoig. similar to gasoline for RPM >900.
  - Most fuels:  $T_{in} < 100^\circ\text{C}$  as speed is reduced  $\Rightarrow$  indicates LTHR (cool flame).
  - Ethanol shows no LTHR!
- Boost has only moderate effect on  $T_{in}$ .



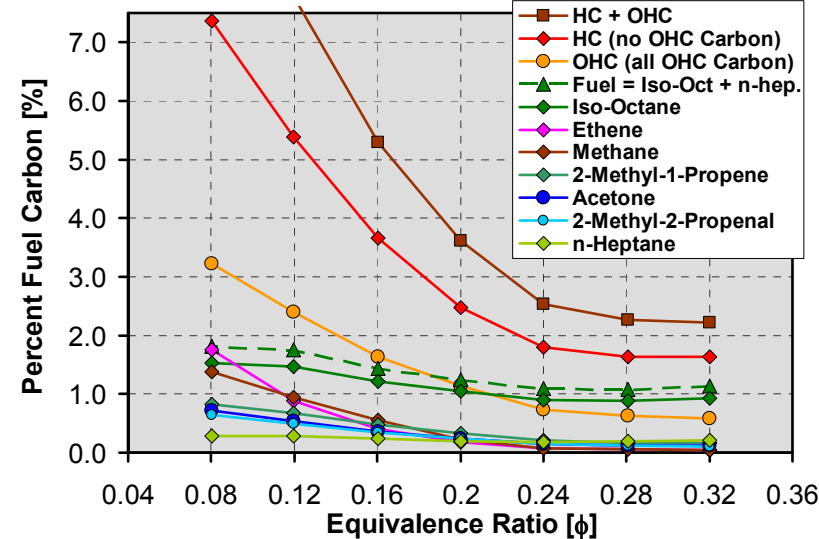
- Performs similarly to gasoline, conds. studied.
- Good potential for HCCI fuel / fuel-component.
- Ethanol is a true single-stage ignition fuel.
  - May offer advantages for control & boosted oper.



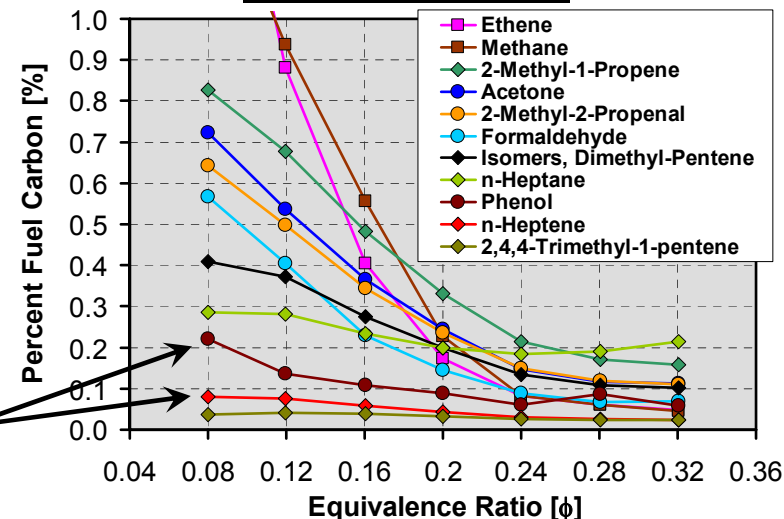
# Detailed Exhaust Speciation – PRF80

- Joint project with LLNL  $\Rightarrow$  spec. analysis.  
Sandia  $\Rightarrow$  engine op. & data interpretation.
- Conducted fueling( $\phi$ )-sweep & data for near-misfire conditions. Results provide:
  - Data for aftertreatment & model validation.
  - Improved understanding of combust. process.
- PRF80 is a 2-stage ignition fuel at conditions studied.  $\Rightarrow$  Affects emissions compared to iso-octane & gasoline.
  - OHC fraction is greater for PRF80.
  - Unreacted-fuel fraction is much lower.  
 $\Rightarrow$  Cool-flame reactions incr. fuel breakdown.
- Ratio of *n*-heptane / iso-octane is 19-23%, compared to 25% in fuel.
  - *n*-Heptane breaks down more readily, but it induces substantial iso-octane breakdown.
- Relatively high conc. of *n*-Heptene & Phenol  
 $\Rightarrow$  former due to *n*-Heptane reactions.

**Main Species**



**Selected Species**



# Future Work

- Complete investigation of intake boost for extending the high-load limit of gasoline-fueled HCCI at a representative speed, 1200 rpm (FY09).
- (FY10) Expand boost study to include a range of higher engine speeds, boost levels, and back-pressures for realistic turbo-charger efficiencies.
  - Two-stage fuels to be done as part of Chevron-funded project.
- Extend TS study: 1) improve diagnostic S/N & optical setup, 2) investigate methods of increasing TS, and 3) determine cause of flows producing TS.
  - Collaborate with J. Oefelein to apply LES modeling  $\Rightarrow$  mechanism / enhancement.
- Additional ethanol studies over a wide range of operating parameters: EGR, load, & boost to high levels  $\Rightarrow$  with M. Sjöberg, Adv. SI-Fuels Lab.
- Complete exhaust-speciation analysis for 2-stage ignition fuel, PRF80, and compare with single-stage ignition fuels, gasoline and iso-oct.  $\Rightarrow$  with LLNL.
  - Analyze emiss. species for near misfire with single- and two-stage ignition fuels.
- Continue to collaborate with LLNL on improving chemical-kinetic mechanisms and on CFD/kinetic modeling.

# Summary

- Quantitative temperature-map images show that thermal strat. (TS) develops progressively during latter compression stroke  $\Rightarrow$  throughout charge.
- Data indicate that TS results from wall-heat transfer and convection.
  - Future work will focus on understanding the mechanism for bulk-gas TS, and potential methods for increasing it to increase the high-load limit of HCCI.
- EGR substantially improves boosted HCCI operation with gasoline fuel.
  - Achieved 16.3 bar IMEP<sub>g</sub>, Ind. Thermal-Eff. = 47%, no Knock & no NO<sub>x</sub> or PM.
  - Near high-load limit for conventional diesel. Shows significant potential for extending HCCI range – full time HCCI?
- Ethanol is a promising HCCI fuel. Performance is generally similar to gasoline, but no low-temp. (“cool-flame”) chemistry.
  - Possible advantages for control and for boosted op.  $\Rightarrow$  additional studies req’d.
- Detailed exhaust speciation of a two-stage ig. fuel (PRF80), shows significantly different behavior from single-stage fuels, iso-octane & gasoline.
  - More breakdown of fuel & fuel-like species to smaller species, higher OHC fract.