

Caterpillar Engine Technologies



An Engine System Approach to Exhaust Waste Heat Recovery

Principal Investigator: Richard W. Kruiswyk Caterpillar Inc. DOE Contract: DE-FC26-05NT42423 DOE Technology Manager: John Fairbanks NETL Project Manager: Carl Maronde

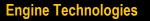
DEER Conference Detroit, MI August 6, 2008



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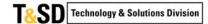


- Overview Engine Thermal Efficiency and Waste Heat Recovery
- EWHR Program Objectives, Timeline, Scope
- Technical Developments
- Summary and Conclusions

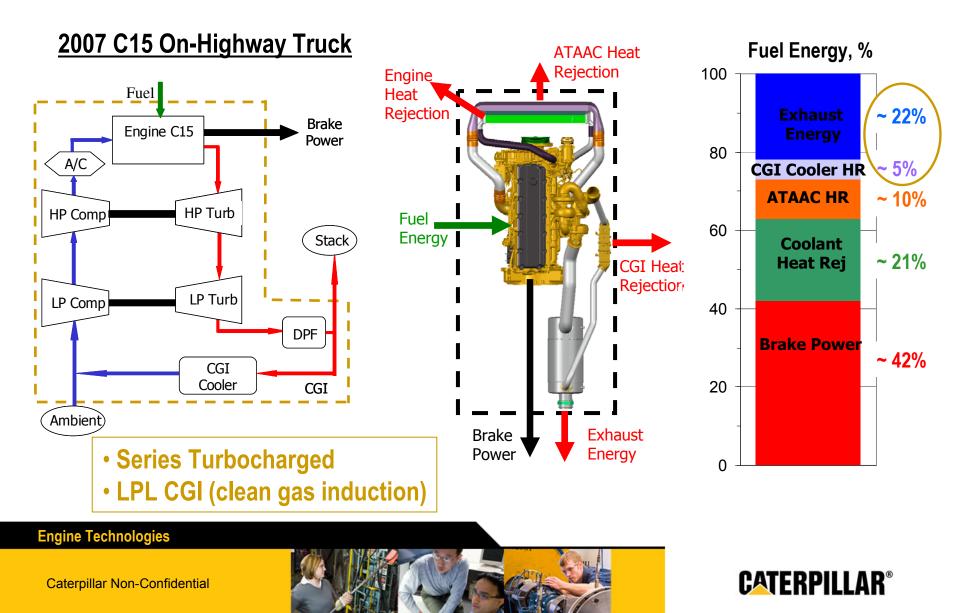






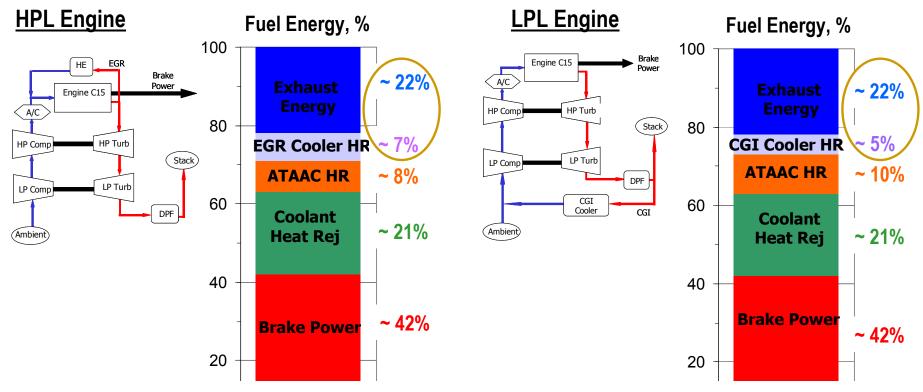


Diesel Engine Thermal Efficiency





Diesel Engine Thermal Efficiency & Exhaust Waste Heat



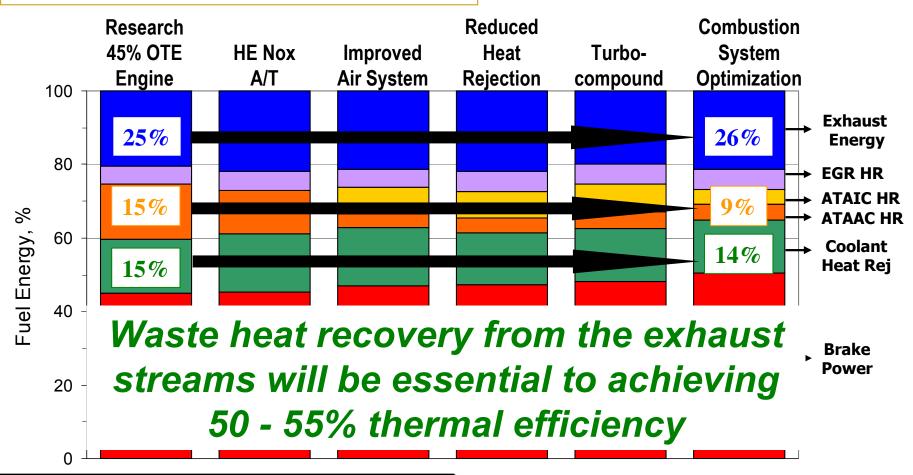
Roughly half of the lost fuel energy is carried in the exhaust streams





Diesel Engine Thermal Efficiency & Exhaust Waste Heat

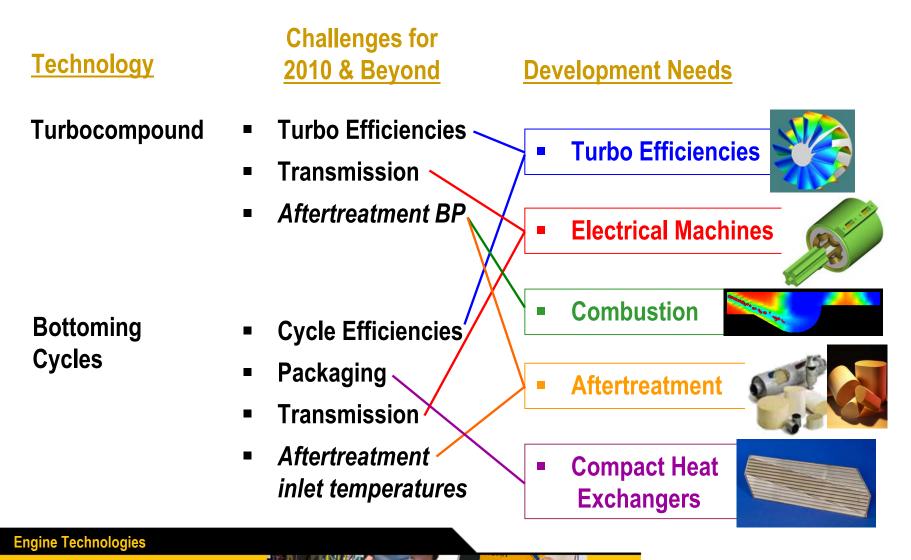
HTCD 50% OTE Program – Analysis Results





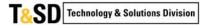


Challenges to Efficient Exhaust Waste Heat Recovery

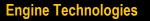


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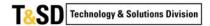


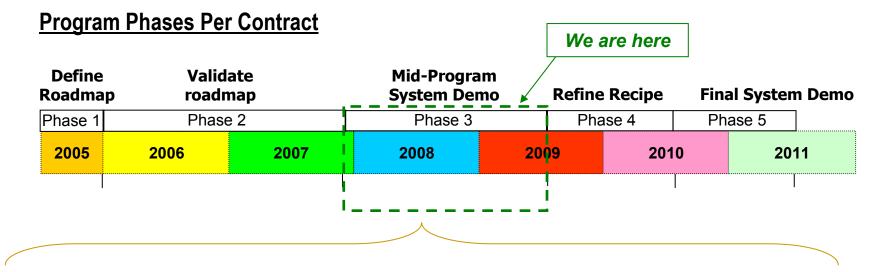
Develop components, technologies, and methods to recover energy lost in the *exhaust processes* of an internal combustion engine and utilize that energy to improve engine thermal efficiency by 10% (i.e. from ~ 42% to ~46% thermal efficiency)

- □ No increase in emissions
- □ No reduction in power density
- Compatible with anticipated aftertreatment









Phase 3 Objective: Demonstration of significant progress (+ 5-10%) in system thermal efficiency improvement via testing/analysis of prototype components. This will include an on-engine system demonstration of prototype hardware.



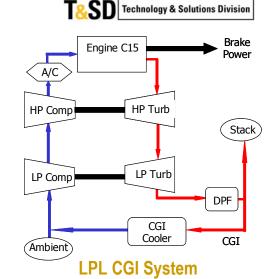


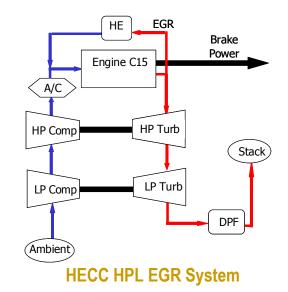


- Solutions compatible w/ production engines transferable technologies
 - Baseline 2007 C15 on-highway engine with LPL CGI system

- Solutions compatible with future engines technologies for HECC solutions
 - HPL or HPL-LPL advantageous in some scenarios

Program scope revised to include investigation of LPL & HPL solutions





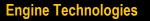


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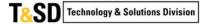
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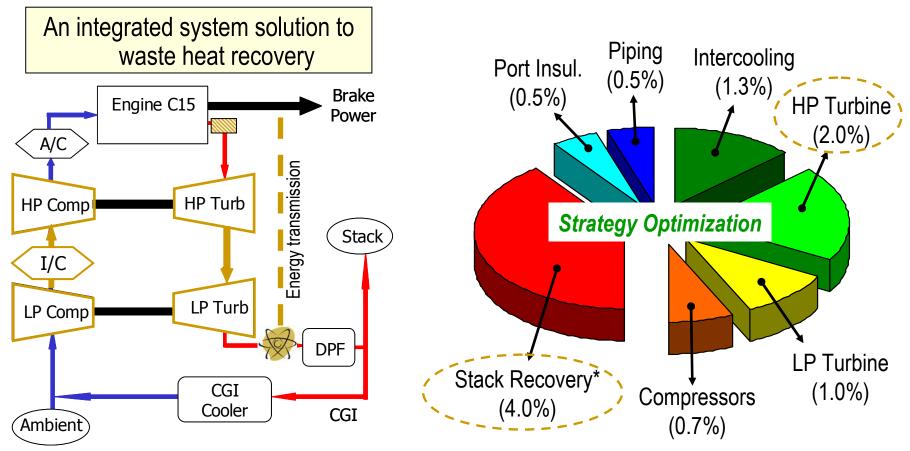








Technical Developments – Proposed System

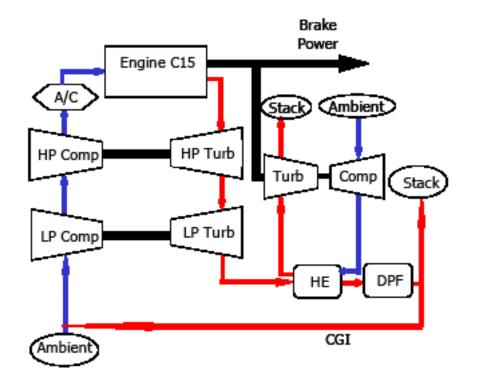


* Turbocompound or bottoming cycle: supplements engine power via electrical or mechanical connection to flywheel

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Early in program:

 2010 A/T backpressure levels not known with certainty

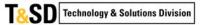
 Turbocompound known to be negatively affected by backpressure

Forced consideration of "backpressure insensitive" solutions



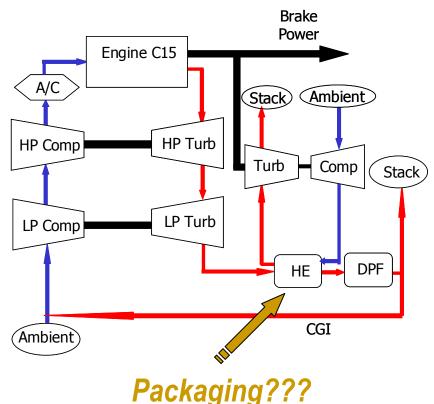
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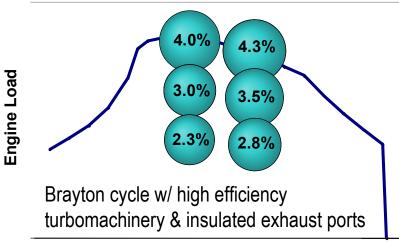


Phase 1 Proposed Solution

Open Air Brayton Cycle



Phase 2 Engine Simulations BSFC Benefit of Brayton on C15



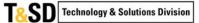
Engine Speed

Performance Objective CAN be met with Brayton Cycle Solution and supporting technologies

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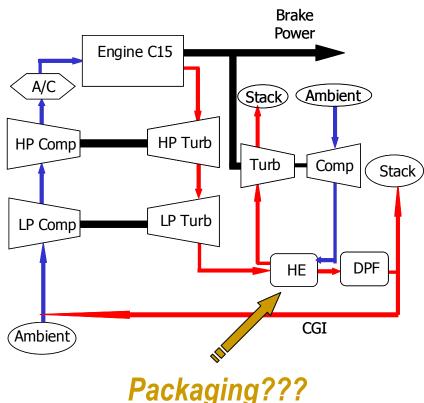
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Phase 1 Proposed Solution

Open Air Brayton Cycle



Phase 2 Heat Exchanger Technology Evaluations

> Primary Surface technology



Caterpillar technology 2 ft³ core vol.

Microchannel manufacturing technology

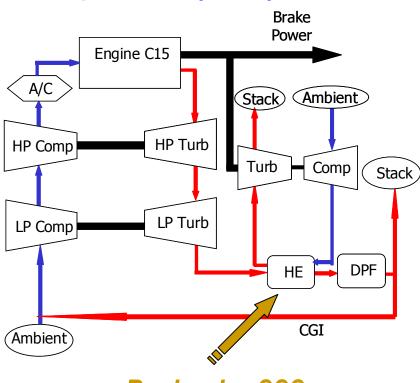
Preliminary estimate: ~ 0.5 - 1 ft³ core Concept design: 1.5 ft³ core vol.

At end of Phase 2, no line of sight to a HE core smaller than 1.5 – 2.0 ft³

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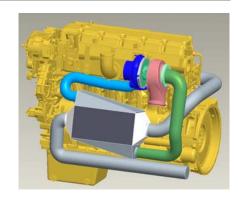


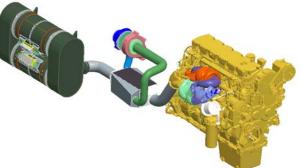
Phase 1 Proposed Solution Open Air Brayton Cycle



Packaging???

Phase 2 Packaging Studies with 1 ft³ heat exchanger core





Even at 1 ft³ core volume, system packaging is challenge

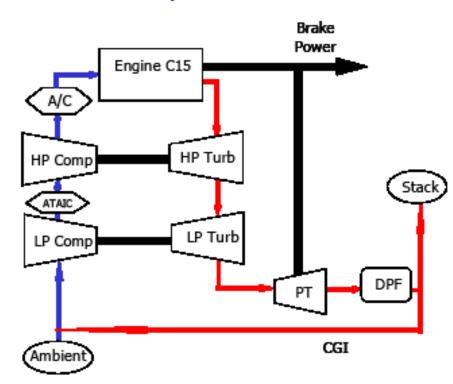
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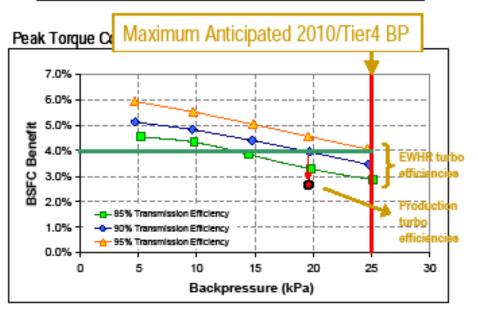




Phase 3 Proposed Solution Turbocompound - Revisited



Phase 3 Engine Simulation Results BSFC Benefit of Turbocompound on C15



Performance Objective CAN be met with turbocompound utilizing <u>high efficiency</u> turbomachinery

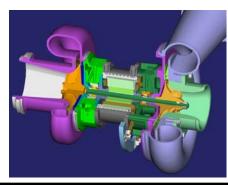
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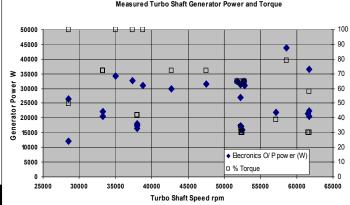
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Electric Turbocompound

- Gen1 Caterpillar / DOE 2001-2004 Development Program
 - Approach Turbocompound function integrated into engine turbo
 - Specification 40kw @ 60krpm in generating mode
 - Results
 - Excellent stability of rotor / shaft system
 - Generated 44kw at 59krpm
 - Turbo generator short of efficiency target
 - Heating problems powers above 15kw limited to 30-60sec runtime

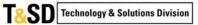






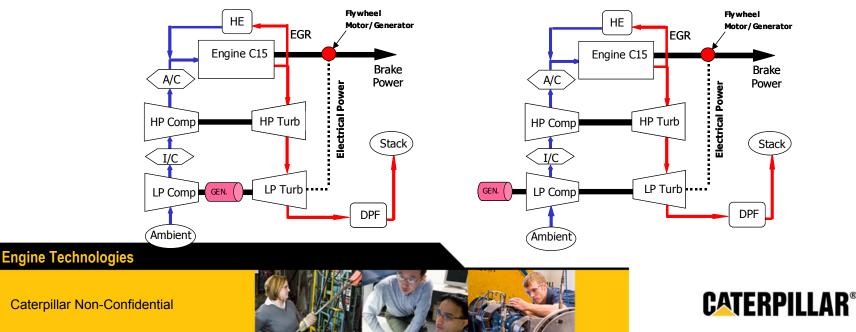
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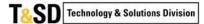


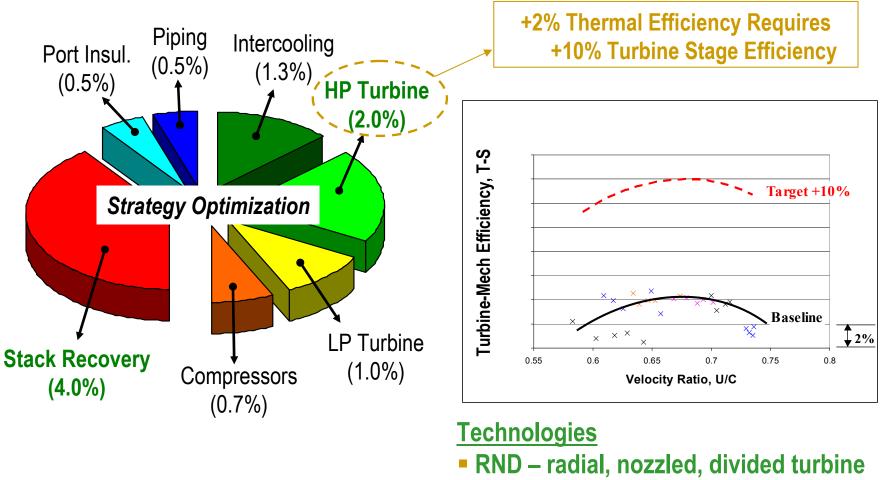
Electric Turbocompound

- Gen2 Concept design collaboration with consultant
 - Specification: 20-25kw power generation
 - Two design options to be evaluated
 - 'Between-the-wheels' option
 - 'In-front-of-compressor' option
 - Thermal, stress, rotordynamic analysis to evaluate options
 - Scheduled completion: 30Oct08





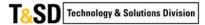


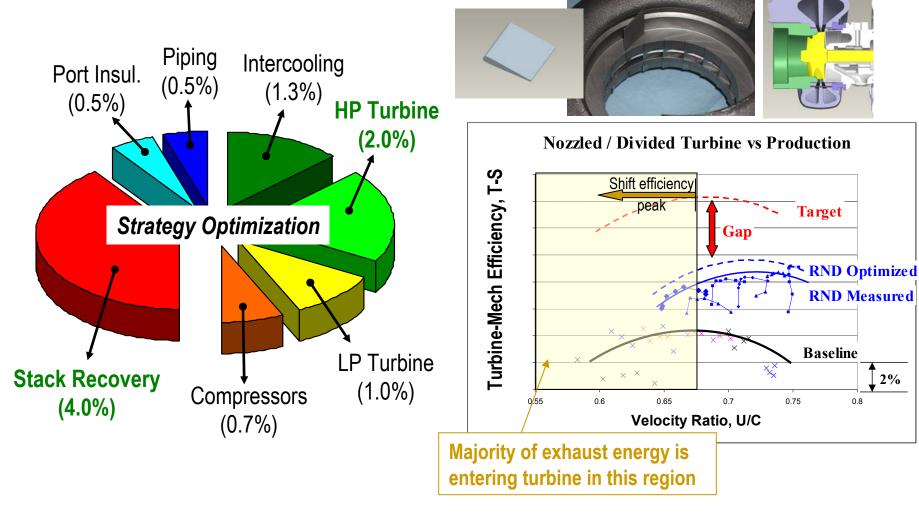


Mixed Flow Turbine

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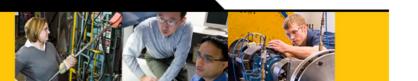






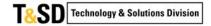
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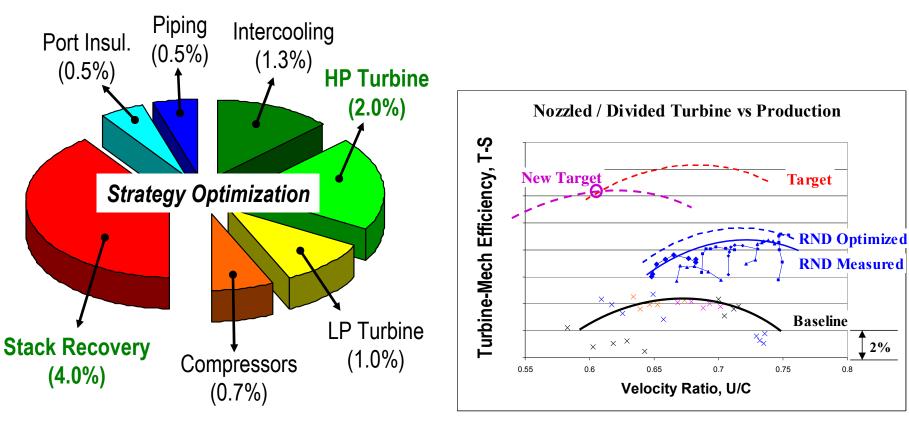
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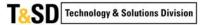




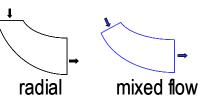
Shifting peak efficiency – mixed flow turbine



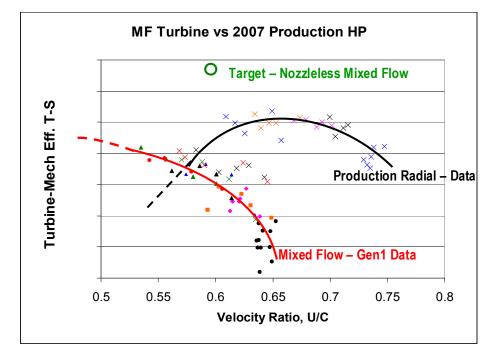




Mixed Flow Turbine



Gen1 wheel designed / procured / tested







Gen1 missed target performance:

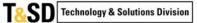
- Over-aggressive design
 - Desire to clearly demo efficiency shift
- First use of codes for mixed flow
 - Calibration required

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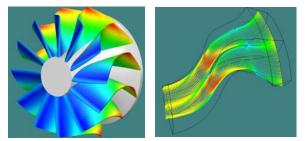
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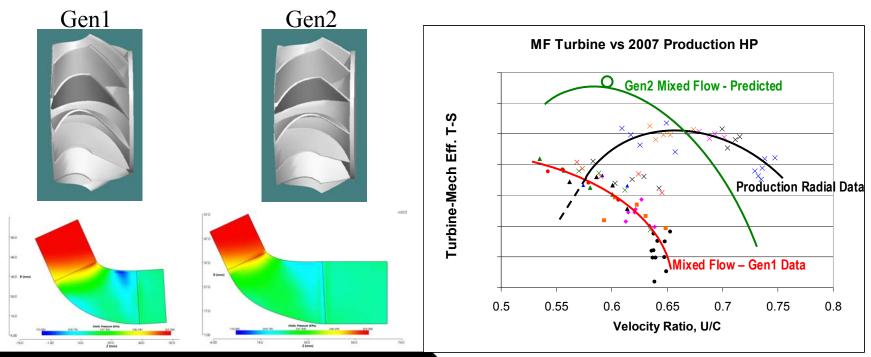


Mixed Flow Turbine

- Gen2 wheel designed / analyzed
 - Gen1 lessons used to optimize design



- 6-8% improvement in efficiency predicted vs Gen1 over range of interest









Technical Developments – HECC HPL EGR Waste Heat Recovery

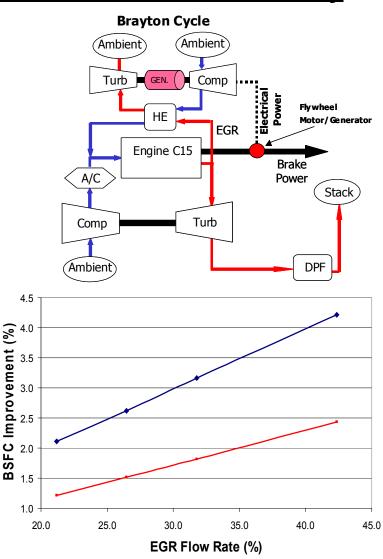
Brayton Cycle

- System Capability Evaluation
- Assumptions:
 - Brayton turbo efficiencies 80%
 - Single stage for packaging
 - Heat exchanger: 90% effectiveness
 - Transmission Efficiency: 90%
- Packaging: Excellent
 - Heat Exchanger: 0.25 ft³ core
 - Turbo:

- ~ 2" compressor
- Performance: Low-Moderate
 - At 20 40% EGR:
 - + ~2 4% w/ 80% turbomachinery
 - + ~1.5 2.5% w/ 74% turbomachinery







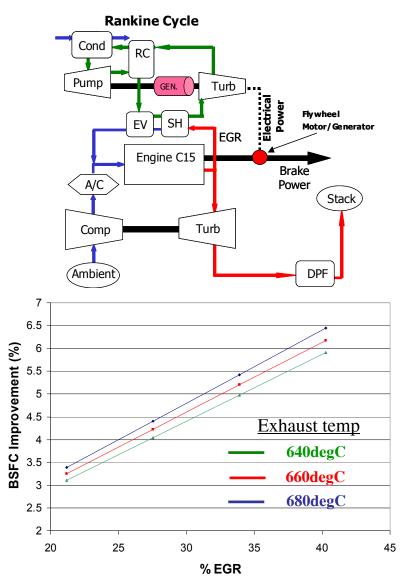
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Technical Developments – HECC HPL EGR Waste Heat Recovery

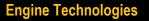
Rankine Cycle

- System Capability Evaluation
- Assumptions:
 - Turbine efficiency 80%
 - Pump efficiency 65%
 - R245fa working fluid
 - Transmission Efficiency: 90%
- Packaging: Challenge
 - Multiple heat exchangers
- Performance: Moderate-Good
 - + 3-6% depending on EGR rates





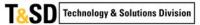
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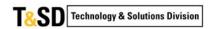


- Significant progress made toward program objectives:
 - With LPL CGI, turbocompound + high efficiency turbos provides +4% BSFC w/ 20kPa aftertreatment backpressure.
 - Progress on supporting technologies, especially turbine technologies, suggests performance target can be met
- Waste heat recovery for future HECC engine solutions prompts consideration of HPL and HPL-LPL EGR configurations
 - Turbocompound still effective for stack recovery. Benefit dependent on aftertreatment backpressure, required EGR rates
 - Brayton cycle offers moderate benefit for HPL loop heat recovery
 - Rankine cycle offers significant benefit for HPL loop heat recovery



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Caterpillar Thanks:

Honeywell

Turbomachinery design consulting, component procurement and integration.

ConceptsNREC

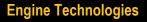
Turbomachinery design consulting and optimization.

Turbo Solutions

Turbomachinery design consulting and optimization.

Barber-Nichols Inc.

Electric turbocompound design consulting



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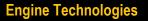
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Caterpillar Thanks:

- Department of Energy
 - Gurpreet Singh
 - John Fairbanks
- DOE National Energy Technology Laboratory
 - Ralph Nine
 - Carl Maronde









Disclaimer

The work described in this presentation, conducted under the Caterpillar / DOE cooperative research agreement, was conducted by the Technology and Solutions Division (T&SD) of Caterpillar Inc. The cooperative research described in the presentation was done to evaluate proof-of-concept for technologies that meet EPA 2010 on-highway emissions with the potential to improve peak brake thermal efficiency by 10%. Cursory consideration was given to which technologies may have some ability to be commercialized by the engine divisions of Caterpillar which have commercialization responsibility. The process to validate technologies as commercially viable was not in the scope of the program, nor was it undertaken. Commercialization aspects such as cost/benefit analysis, reliability, durability, serviceability and packaging across multiple applications were only considered at a cursory level. Until such analysis is completed, any attempt to imply commercial viability as a result of the material in this presentation is not justified.

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